Analysis of Heat Transfer in the VVER-1200 Reactor's Heat Channel

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Abstract: In early 2015, the Government of Vietname has decided to choose VVER-1200 Russian-made technology for building at the Nuclear Power Plant in Ninh Thuan 1, this is the advanced reactor generation $III +$ and the only one has been completed for the first time in the world in August 2016. Vietnam is facing a major challenge, which is how to ensure the acquired technology transfer process, then the safe operation of thisunit.

This article analyzes some of the heat changes occur in reactor when there are changes of the heatflux. This is an issue directly related to the workof predictingincidents and give ways to fix the problem when the plant is in conditions such as startup, normal and abnormaloperations.

For analysis, the authors used CFD methods, this is a very modern method and have high reliability. The results received have fit well when compared with the safety analysis report of Rosatom published.

Keywords: Reactor thermalhydraulics, VVER-1200, CFD.

1. Introduction

The core of VVER-1200 reactor is designed with due consideration of the TOR for the reactor plant of NPP-2006, which stipulates considerable increase of the parameters determining the performance of an nuclear power plant– performance factor and availability factor of the Unit as compared to the commercial VVER-1000 reactor. In particular, it is necessary to increase the thermal power to 3200 MW, to provide for 12 months long operation between refuelling taking into account planned outage for refuelling. The basic fuel cycle is considered to have the cycle length about 340 EFFD, maximum fuel burnup in fuel assemblies is expected to be up to 70MW⋅day/kgU[4].

Many of the technical and design solutions, used in the design of the core VVER-1200. The main solutions providing for increasing the amount of fuel in the core are asfollows:

Elongation of the fuelcolumn.

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Increasing the external diameter of the fuelpellet. Reducing the centrelinehole.

Fig. 1. Fuel assembly and Fuel Rod.

2. Researching methodology

Computational fluid dynamics or CFD is the analysis of systems involving fluid flow, heat transfer and associated phenomena such as chemical reactions by means of computer-based simulation. The CFD method uses the meshing tools to separate the large-sized objects into the small pieces then applies three conservation equations of physics:Newton's second law; Mass conversation and first law of thermodynamics to solve the problems.The technique is very powerful and spans a wide range of industrial and non-industrial application areas [1,7].

We describe the behavior of the fluid in terms of macroscopic properties, such as velocity, pressure, density and temperature, and their space and time derivatives. We consider such a small element of fluid with sides δx , δy and δz . All fluid properties are functions of space and time: $\rho(x, y, z, z)$ *t*), *p*(*x*, *y*, *z*, *t*), *T*(*x*, *y*, *z*, *t*) and *u*(*x*, *y*, *z*, *t*)[1,7].

We have the temperature of the fuel rod cladding outer surface depends on coolant temperature, heat flux value and heat transfer coefficient from fuel rod surface [2,4,6]:

$$
T_{\text{clad}}(z) = T_{\text{w}}(z) + \frac{q^{\text{w}}(z)}{\alpha} \tag{1}
$$

Where: Tclad (z)- Temperature of the fuel rod cladding outer surface, $^{\circ}C$; q''(z)- Heat flux from the fuel rod surface, kW/m². °C; T_w(z)- Coolant temperature, °C; α - Heat transfer coefficient from the fuel rod surface, kW/m^2 . °C.

With the core coolant parameters corresponding to values of normal operating conditions, the heat transfer coefficient under the conditions of forced convection of one-phase subcooled coolant is determined by formulae [8,10]:

$$
\alpha = \frac{\lambda}{d_s} Nu
$$

\n
$$
Nu = \begin{cases}\n3.66; Re < 2300 \\
3.66 \left(\frac{Re}{2300}\right)^{1.565 + 0.272 \ln(Pr)}; 2300 \le Re < 10^4 \\
0.023 Re^{0.8} Pr^{0.4}; Re \ge 10^4\n\end{cases}
$$
\n(2)

Where: Nu-Nusselt number; Re-Reynolds number; Pr-Prandtl number; λ-heat conductivity,W/m; dg –Diameter, m.

When coolant water flows from lower part to upper part in fuel assembly, it occurs the pressure ps:
 $p_{in} - p_{out} = \Delta p_{inertia} + \Delta p_{acc} + \Delta p_{gravity} + \Delta p_{friction} + \Delta p_{form}$ (3) drops:

$$
p_{in} - p_{out} = \Delta p_{inertia} + \Delta p_{acc} + \Delta p_{gravity} + \Delta p_{friction} + \Delta p_{form}
$$

\nwith:
$$
\Delta p_{inertia} = \sum_{n=1}^{N} \frac{l_n}{A_n} \frac{d\dot{m}}{dt}; \quad \Delta p_{acc} = \frac{\dot{m}^2}{2\rho} \left(\frac{1}{A_N^2} - \frac{1}{A_1^2} \right);
$$

$$
\Delta p_{gravity} = \rho g \left(z_N - z_1 \right); \quad \Delta p_{form} = K \left(\frac{\rho v_{ref}^2}{2} \right); \Delta p_{friction} = \overline{f} \frac{L}{D} \left(\frac{\rho v_{ref}^2}{2} \right)
$$
(3)

Where: ρ-Density, kg/m3; g-Accelerator, m2/s; v-Velocity, m/s; m -Mass flow rate,kg/m; N- number of sections.

3. Results and discussions

We consider the changes of some main parameters under the different values of heat flux, the range of heat flux is from $0.3x10^6$ W/m² to 1.486x10⁶W/m², which is critical heat flux. Coolant water have velocity is 5.4865m/s, temperature is 271.16K and pressure is 16.2MPa. The heat coolant

channel has 4 fuel rods surround and 12 spacer grids inside as shown in Fig. 2. We obtained some results as following below [2, 6, 7]:

We get data from the line and plane in the middle of the heat coolant channel, the results are provided in the table 2 and fig 3, fig 4, fig 5.

Fig 4. The line and plane in middle of heat coolant channel.

Table 2. The obtained results by CFD analysis.

Heat flux	$T_{in}(K)$	$T_{out}(K)$	T_{clad} min (K) T_{clad}		T_{clad} max
(W/m ²)				averaged(K)	(K)
300000	571.16	589.08	571.18	581.81	590.75
400000	571.16	594.70	571.19	585.21	596.84
500000	571.16	599.94	571.20	588.50	602.51
600000	571.16	605.08	571.22	591.71	608.00
700000	571.16	609.90	571.23	594.81	613.09
800000	571.16	614.37	571.24	597.78	617.73
900000	571.16	618.45	571.25	600.62	621.84
1000000	571.16	622.07	571.26	603.30	625.29
1100000	571.16	625.16	571.27	605.80	628.01
1200000	571.16	627.55	571.28	608.10	629.33
1300000	571.16	629.04	571.29	610.12	630.17
1486199	571.16	630.14	571.31	613.23	630.23

Fig 5. Temperature distribution with the change of heat flux.

From the chart, we get the function related between temperature at outlet plane and heat flux: the function related between temperature at outlet plane and heat flu
 $T_{out} = 572.622 + 5.388 \times 10^{-5} q^{n} + 7.478 \times 10^{-12} q^{n^2} - 1.191 \times 10^{-17} q^{n^3}$

Fig 6a. Cladding temperature changes with heat flux.

(4)

Fig 6b. Cladding temperature changes with heat flux.

The function related between maximum cladding temperature and heat flux:
\n
$$
T_{\text{cladMax}} = 571.2 + 6.618 \times 10^{-5} q^{\text{H}} - 8.491 \times 10^{-13} q^{\text{H}} - 1.147 \times 10^{-17} q^{\text{H}} \tag{5}
$$

From all results, we can see that CFD method is useful and powerful tomodel and simulate all fluid processes, including fluid-structure multiphysicsinteractions. The results show that if the heat flux change from $0.3x10^6$ W/m² to1.486x10⁶ W/m², then all safety criteria are ensure. The maximumtemperature offuel rod cladding is 630.23K, which is lower than melting temperature value [6, 7].

In addition, regarding to the velocity change we can see when the heat flux is $0.5x10^6$ W/m² the outlet temperature is 599.94K that is suitable in normal operation conditions of VVER-1200.

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